

Analysis of the SIS100 magnet cooling

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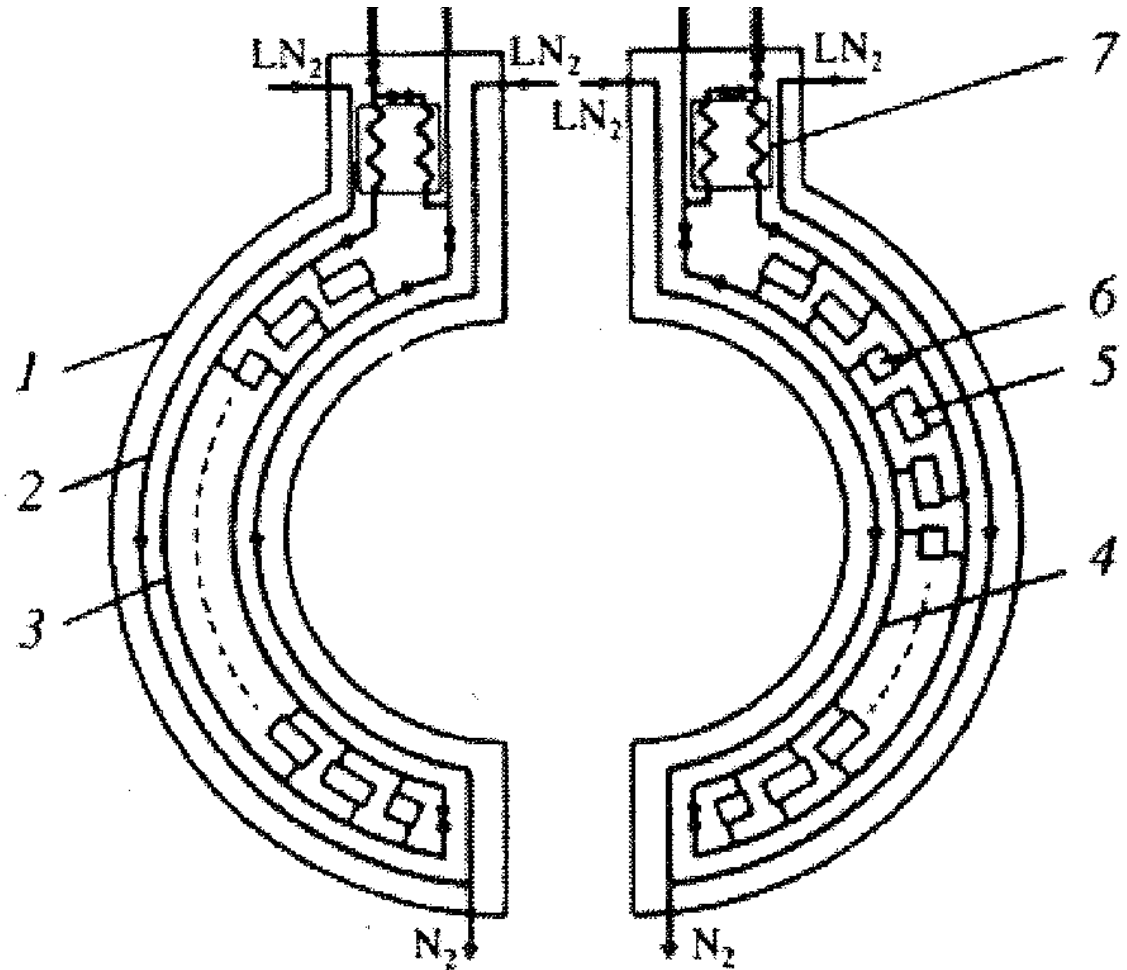
Outline

- **SIS100 cooling concept**
- **Analysis of the single dipole cooling**
- **Arrangements needed for parallel cooling**
- **Conclusions**

Nuclotron Ring Cooling

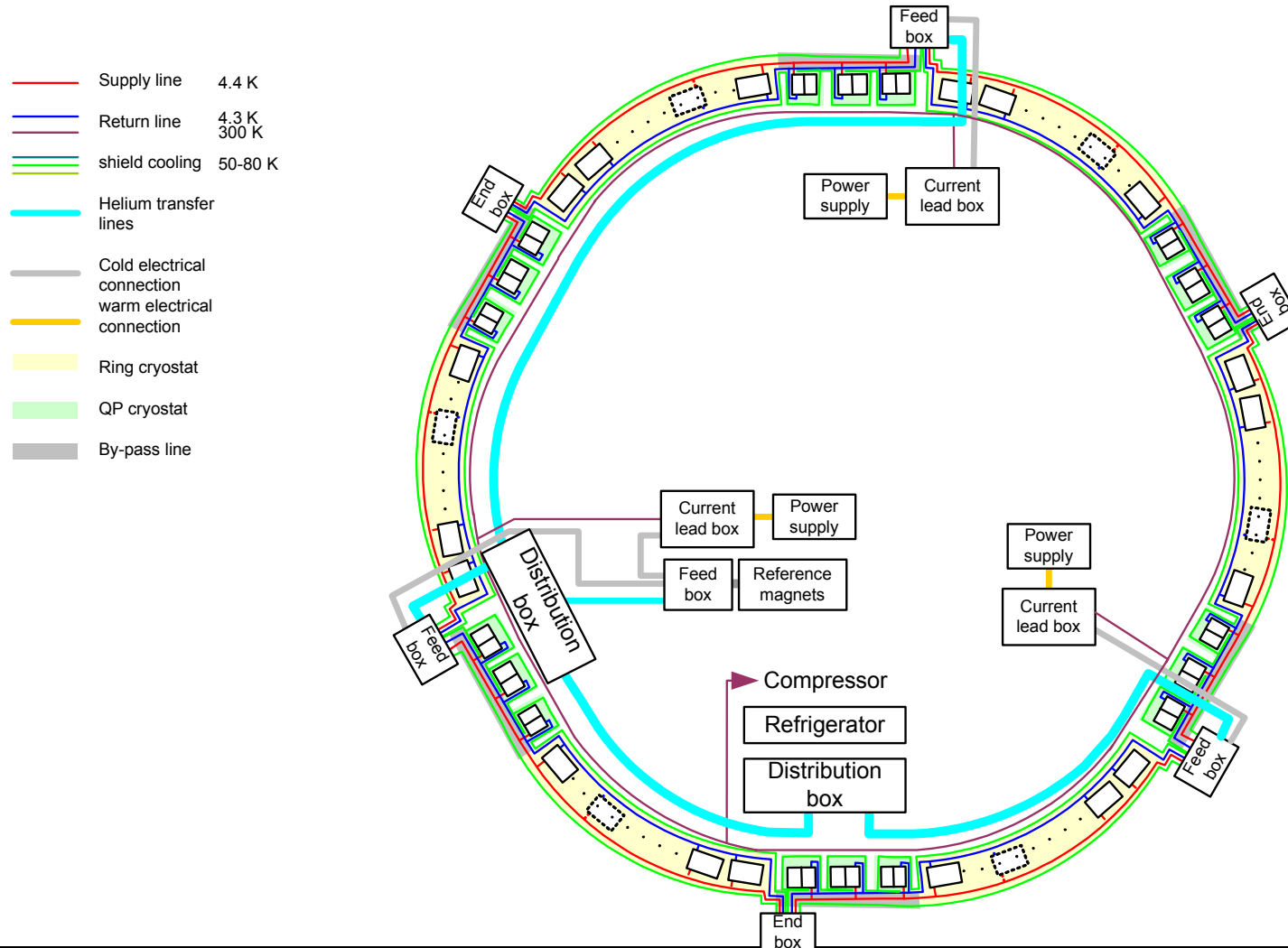
1. vacuum shell
2. nitrogen shield
3. supply header
4. return header
5. dipole magnet
6. quadrupole magnet
7. subcooler

2 x 100 parallel channels

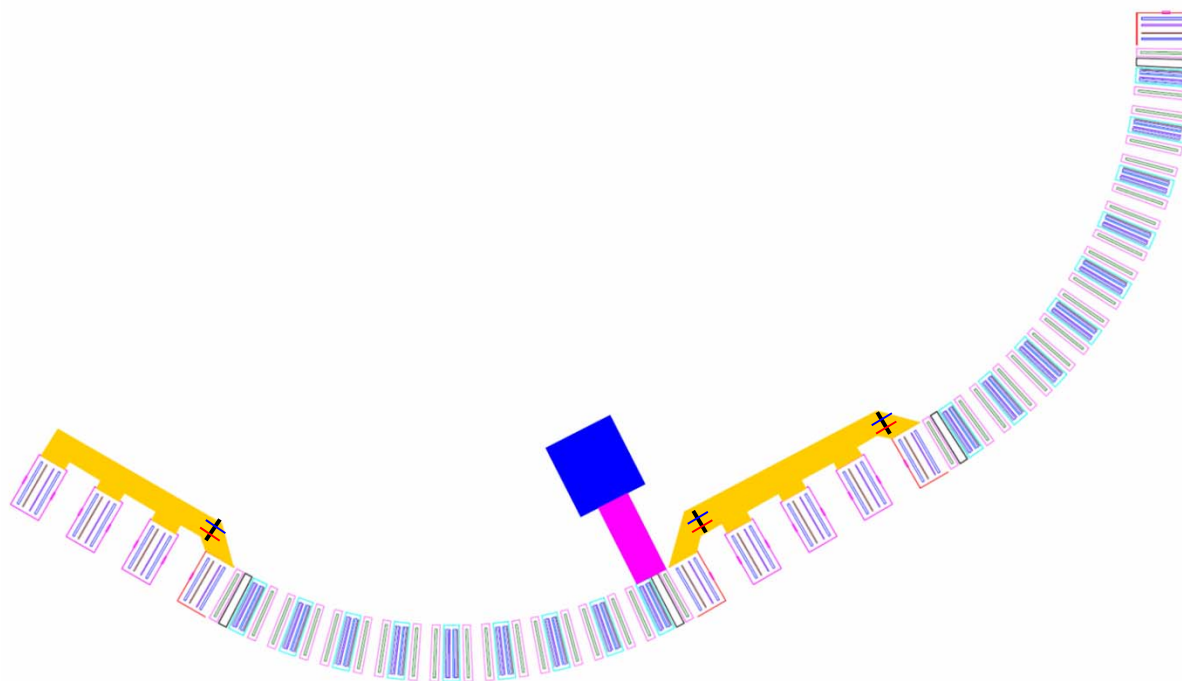
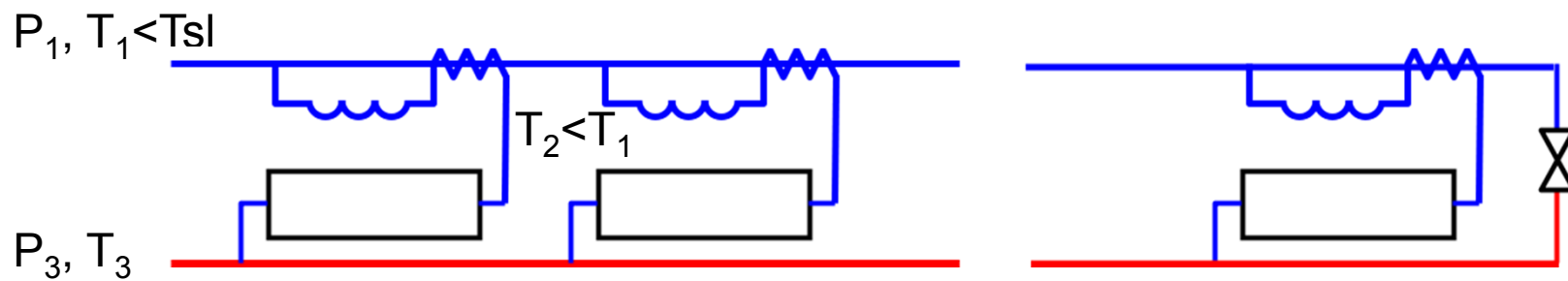


SIS100 Ring Cooling

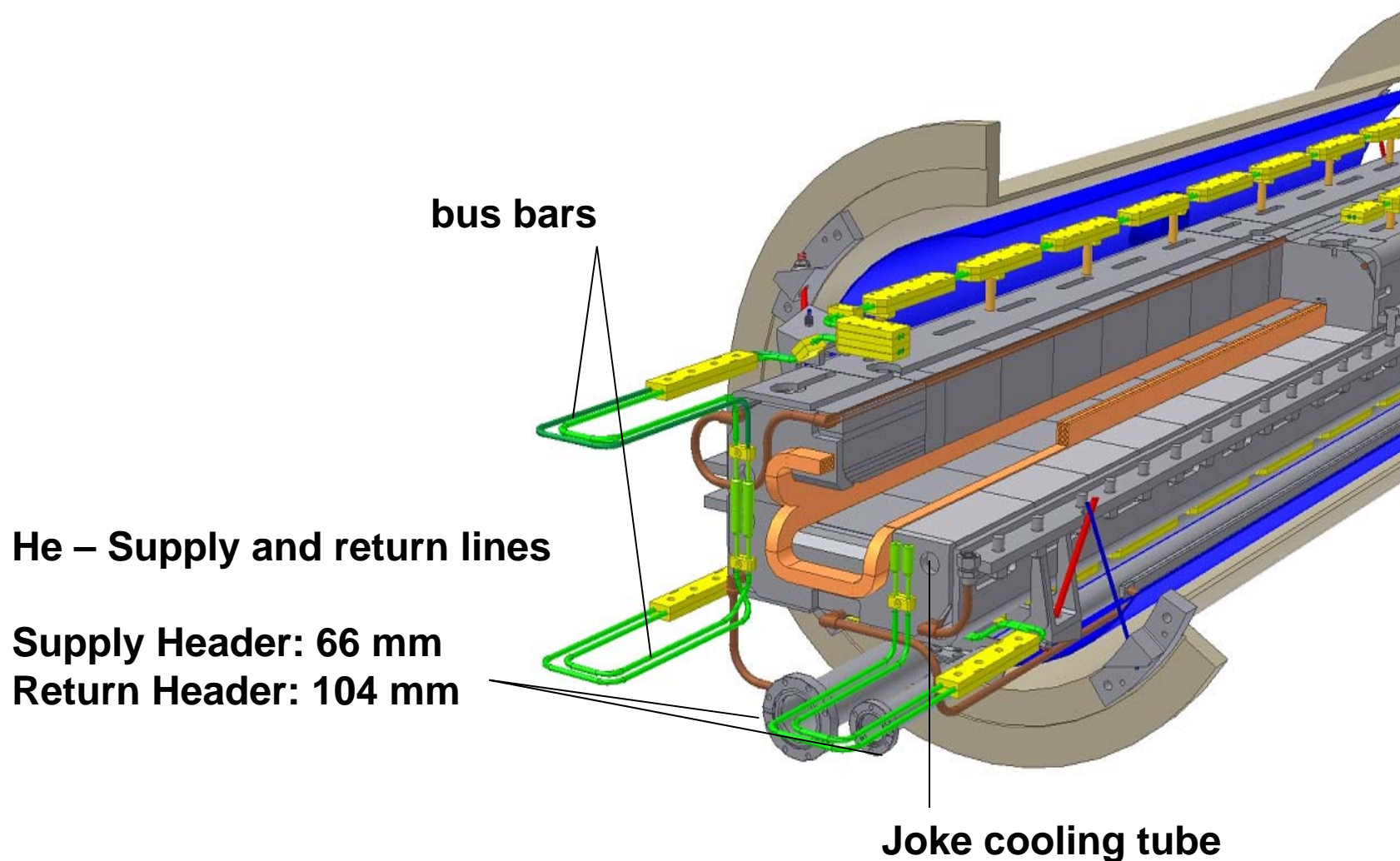
Common Specification for Cryogenic Feedboxes (M.Kauschke, Y.Xiang, Ver. 2010.12.15)



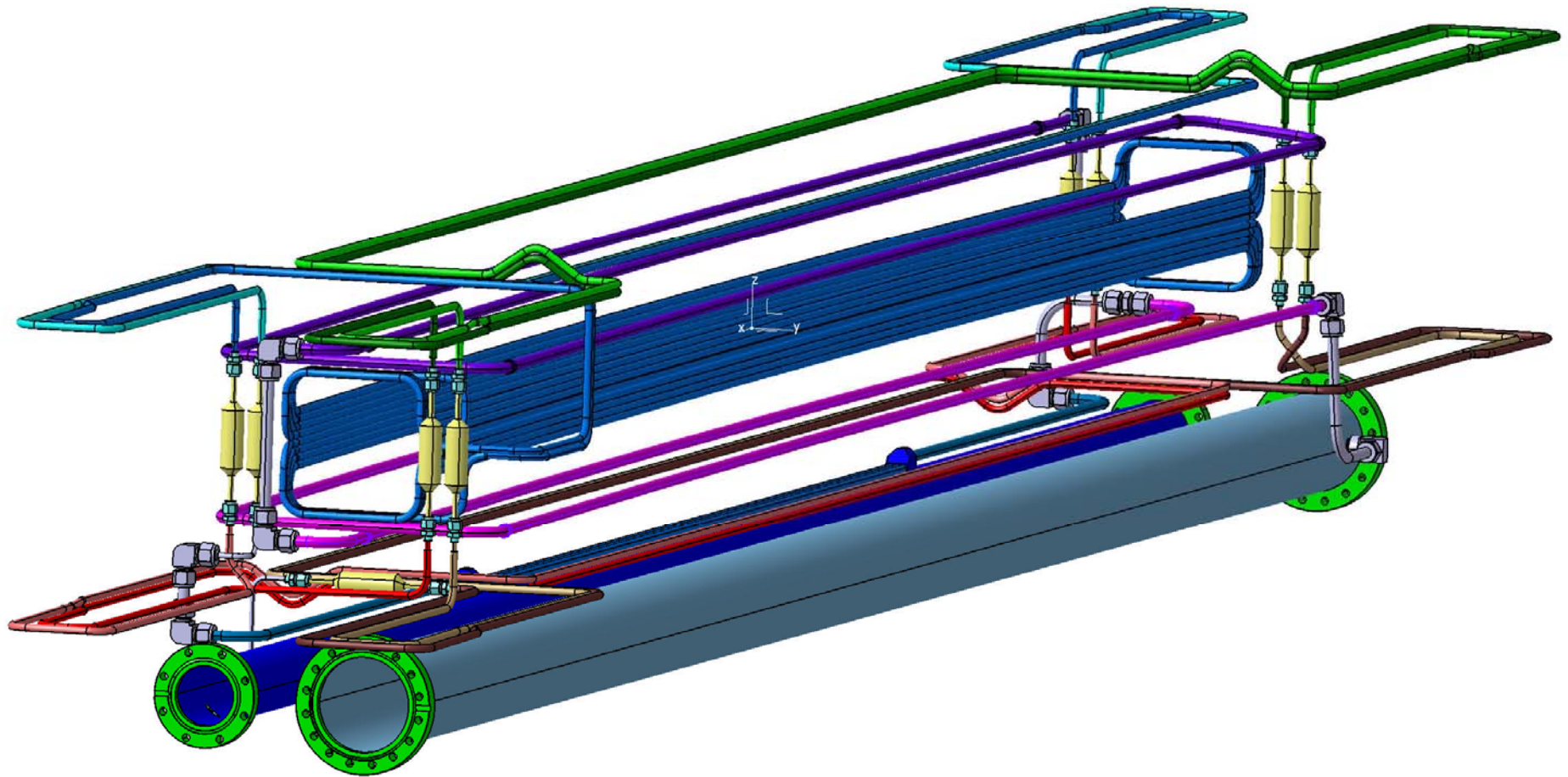
SIS100 Sector



SIS100 dipole cooling

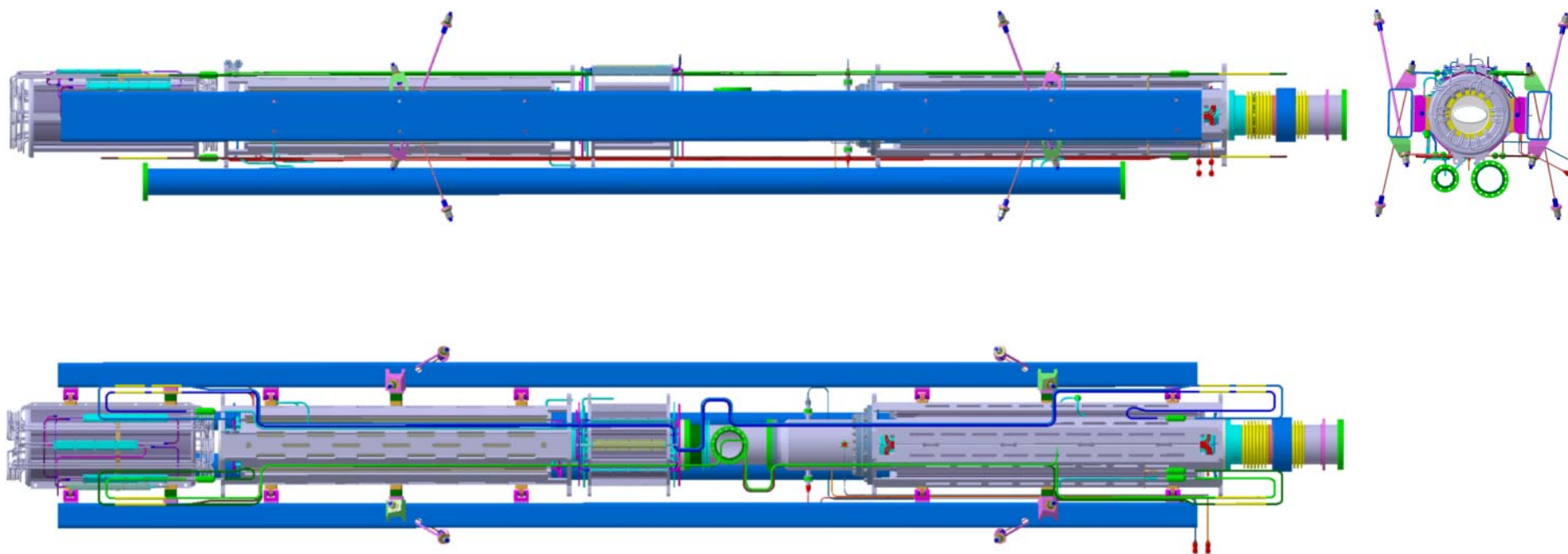


SIS100 dipole cooling

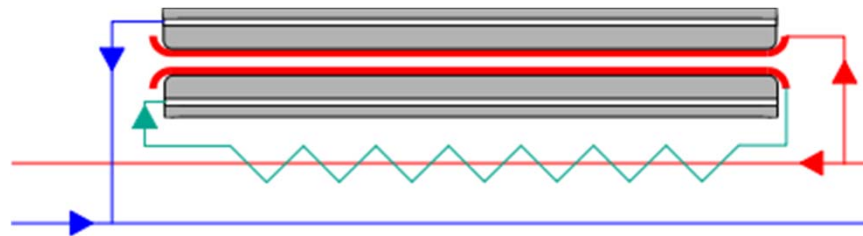


0,312

Quadrupole Doublet



SIS100 dipole cooling



Inlet – sub-cooled helium

$P_{in} = 1.5 \text{ bar}$, $T_{in} = 4.5 \text{ K}$

Coil out:

$P = 1.1 \text{ bar}$, two-phase (4.3 K)

Joke out:

$P = 1.1 \text{ bar}$, two-phase, $x = 0.9 - 1.0$

Heat load:

static: 7 W

dynamic: up to 60 W (triangular cycle)

Mass flow: defined by the total heat load

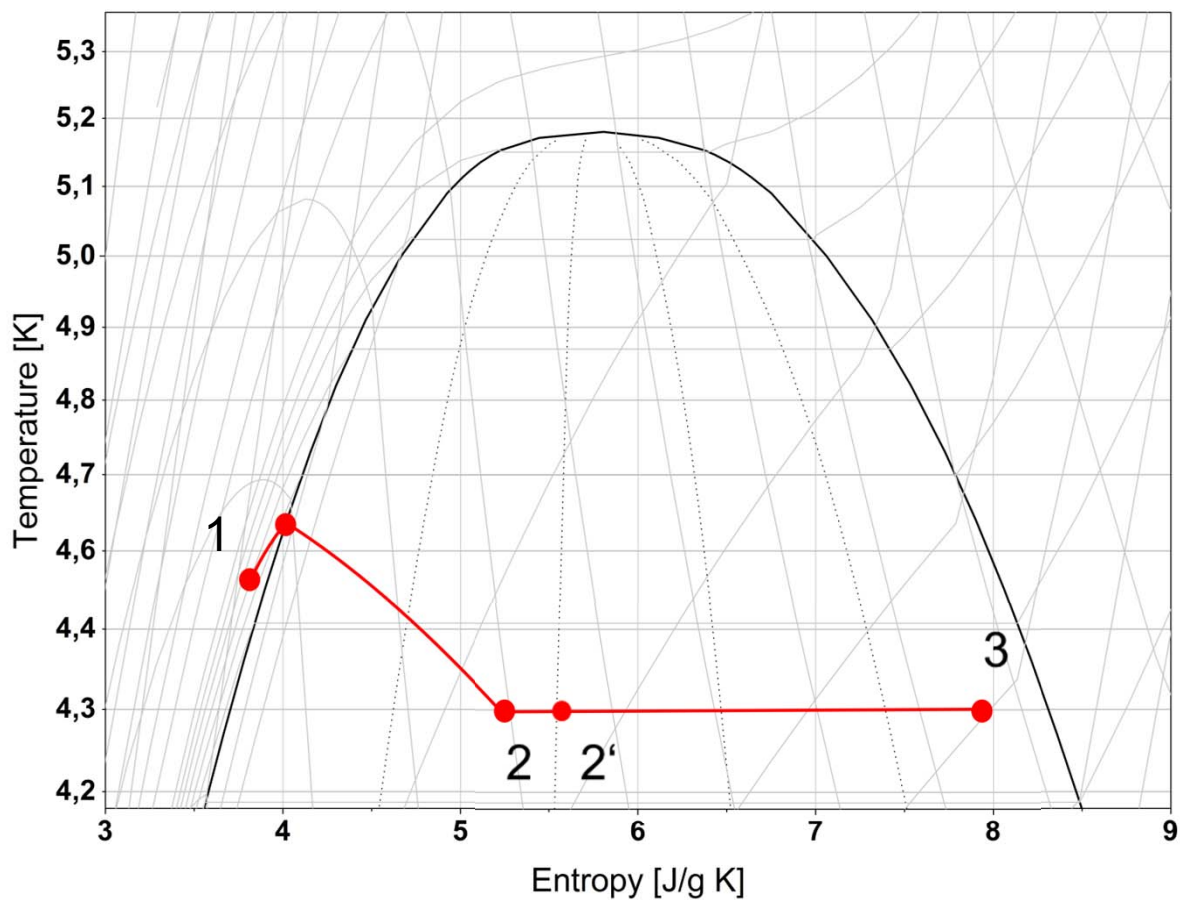
Pressure dprop: defined by the mass flow rate
and hydraulic resistance of cooling channels:

cable inner diameter: $d = 4.7 \text{ mm}$ iron yoke: $d = 10 \text{ mm}$

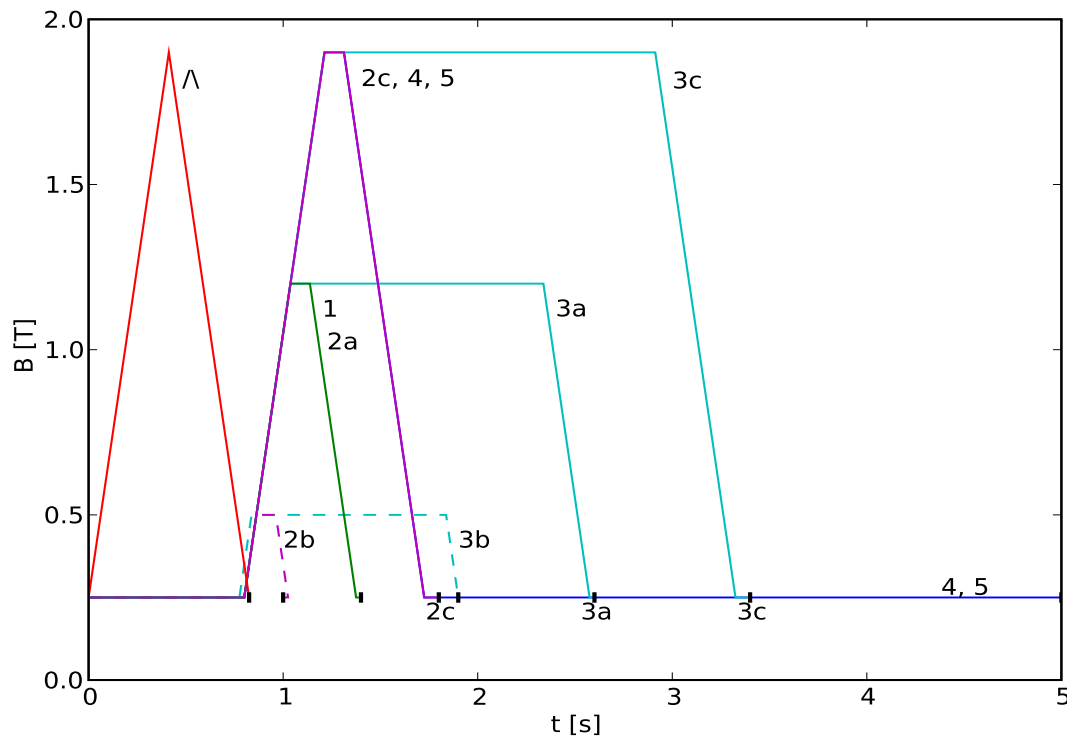
dipole: $L = 54 \text{ m} + 54 \text{ m}$

quadrupole: $L = 34 \text{ m} + 27 \text{ m}$

Single Magnet: T-S Diagram



Operating Modes



Operating cycles

Cycle	B_{\max} [T]	t_f [s]	t_c [s]
1	1.2	0.1	1.4
2a	1.2	0.1	1.4
2b	0.5	0.1	1
2c	1.9	0.1	1.8
3a	1.2	1.3	2.6
3b	0.5	1.0	1.9
3c	1.9	1.7	3.4
4	1.9	0.1	5.0
5	1.9	0.1	5.0
Λ	1.9	0	1.0

Expected Losses

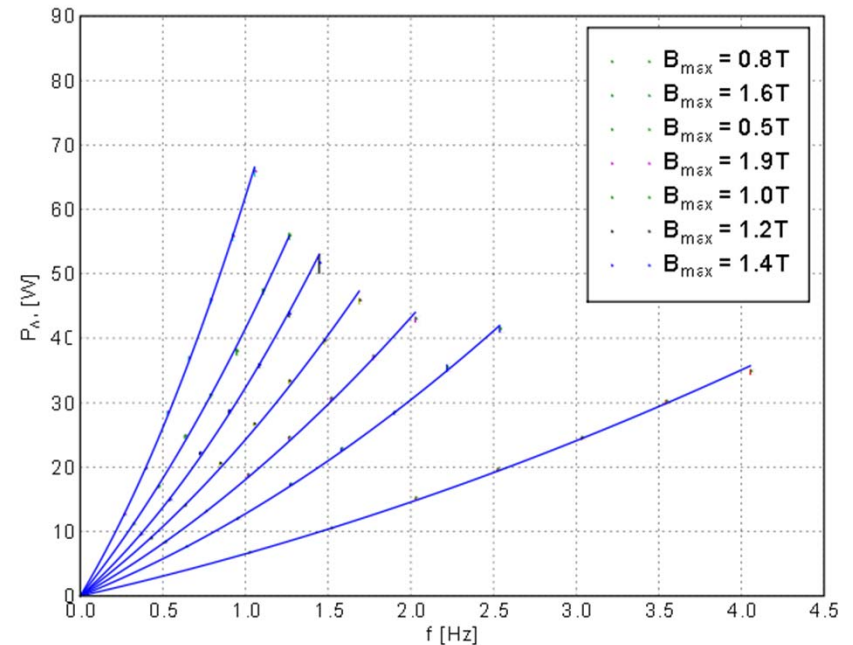
Measurements on dipole prototypes:

dynamic losses:
up to 60 W in the triangular cycle

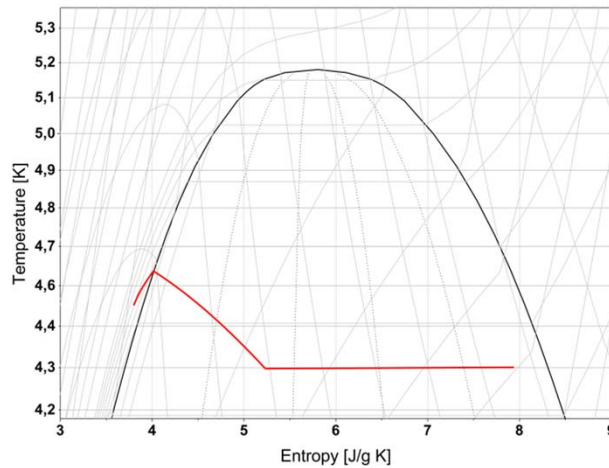
static losses:
up to 10 W

losses in the vacuum chamber
(to be cooled separately):
up to 14 W

Quadrupole unit:
50 % of dipole losses (estimated)



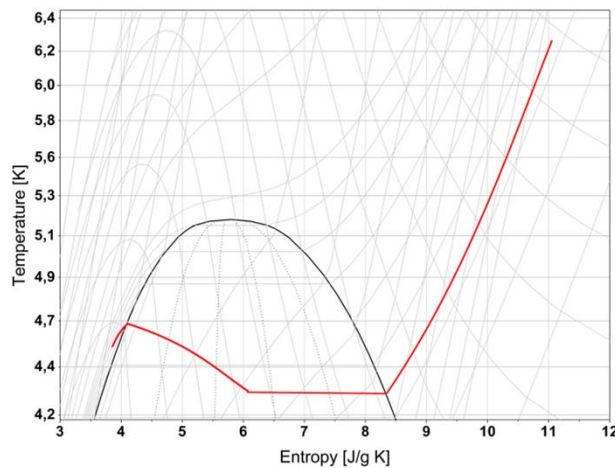
Dipole cooling



Busbar + Coil: 54 m + 54 m
Minor losses: $\xi = 26.0$

$P_1 = 1.6 \text{ bar}$, $T_1 = 4.51 \text{ K}$ (subcooled)
 $P_3 = 1.1 \text{ bar}$, $T_3 = 4.31 \text{ K}$

$P_{\text{total}} = 40 \text{ W}$: $m = 2.4 \text{ g/s}$
 $x_2 = 0.30$
 $x_3 = 0.88$

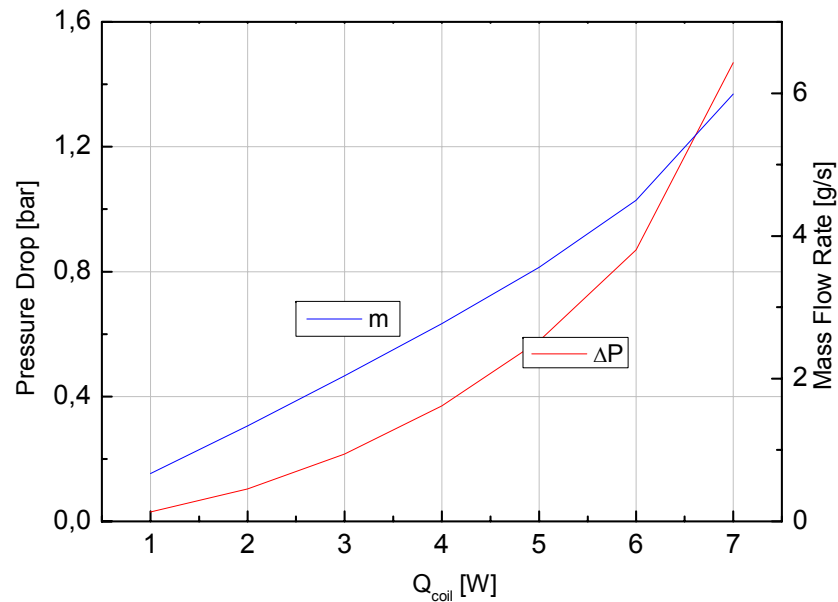


$P_{\text{total}} = 70 \text{ W}$: $m = 2.1 \text{ g/s}$
 $x_2 = 0.56$
 $T_3 = 6.52 \text{ K}$

Two-Phase Cooling

$$Q_{\text{yoke}} = 30 \text{ W}$$

$$T_1 = 4.55 \text{ K}$$

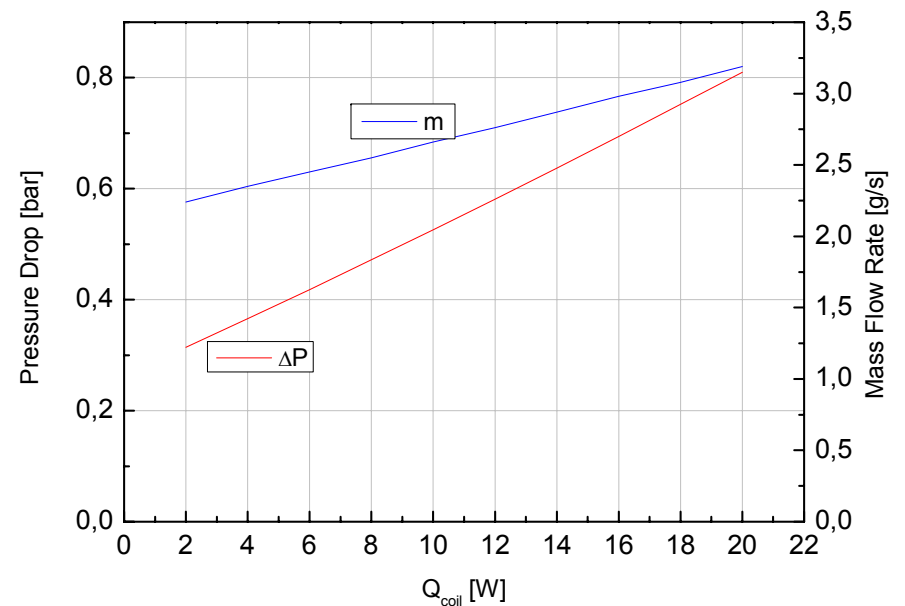


Supercritical Helium:

$$P_2 = 2.3 \text{ bar}$$

$$T_2 = 4.7 \text{ K}$$

$$m = Q_{\text{coil}} / (h_2 - h_1)$$

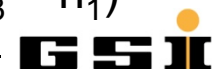


Two-Phase Helium:

$$P_2 = 1.1 \text{ bar}$$

$$X_3 = 1$$

$$m = (Q_{\text{coil}} + Q_{\text{yoke}}) / (h_3 - h_1)$$



Two Phase Flow: Baker Diagram

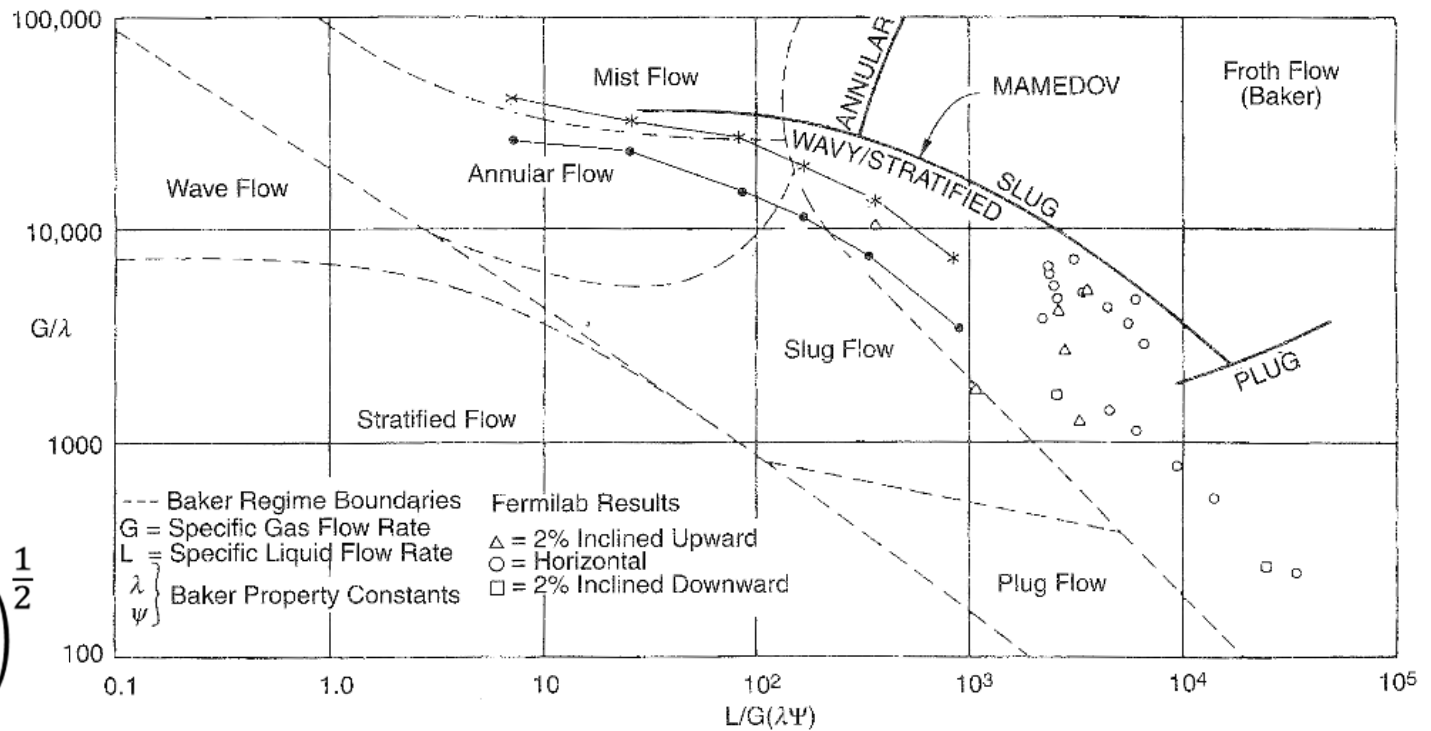
Y axis: gas mass velocity

$$X = \frac{\dot{m}_L}{\dot{m}_G} \lambda \psi$$

$$Y = \frac{\dot{m}_G}{\lambda}$$

$$\lambda = \left(\frac{\rho_G}{\rho_{air}} \frac{\rho_L}{\rho_{water}} \right)^{\frac{1}{2}}$$

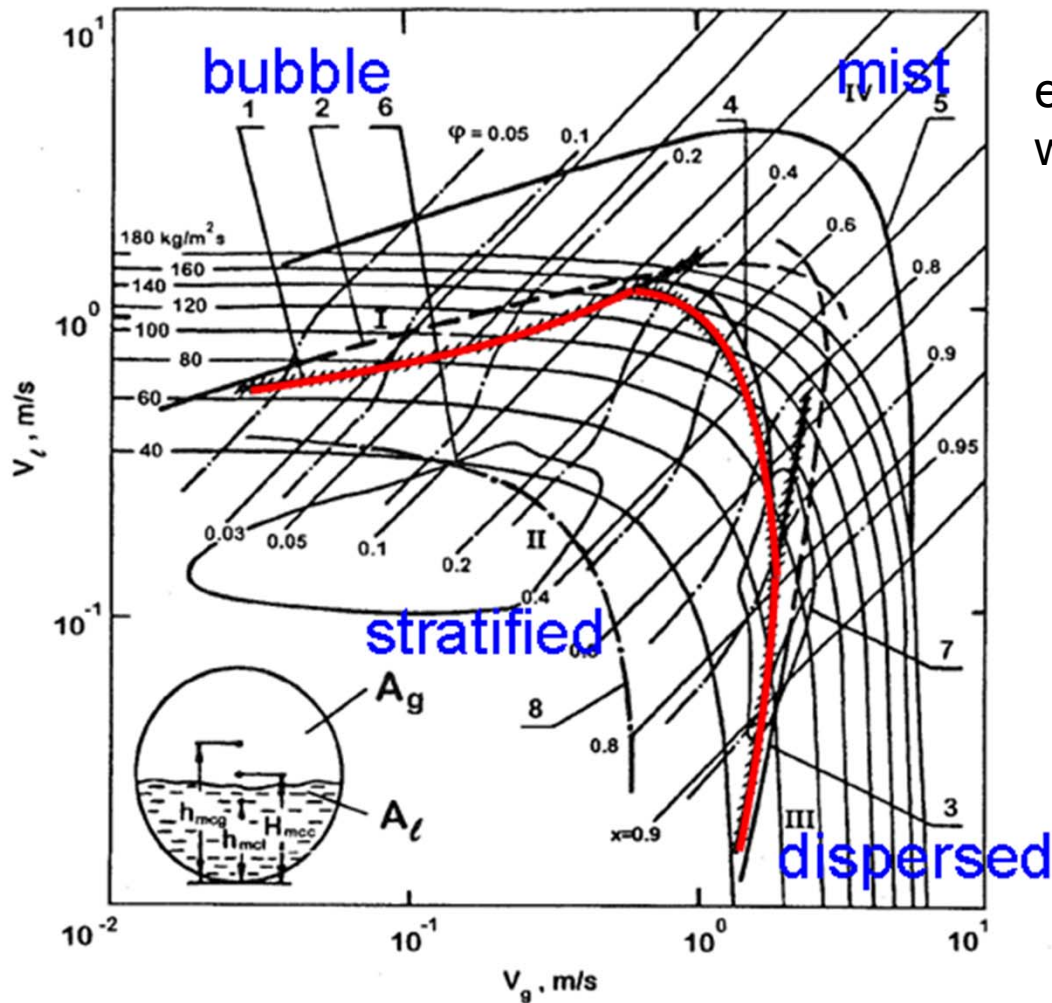
$$\psi = \left(\frac{\sigma_{water}}{\sigma} \right) \left[\left(\frac{\mu_L}{\mu_{water}} \right) \left(\frac{\rho_{water}}{\rho_L} \right)^2 \right]^{\frac{1}{3}}$$



X axis: effective liquid-gas ratio

flow pattern map

Y. Filippov/Cryogenics 39 (1999) 59-68



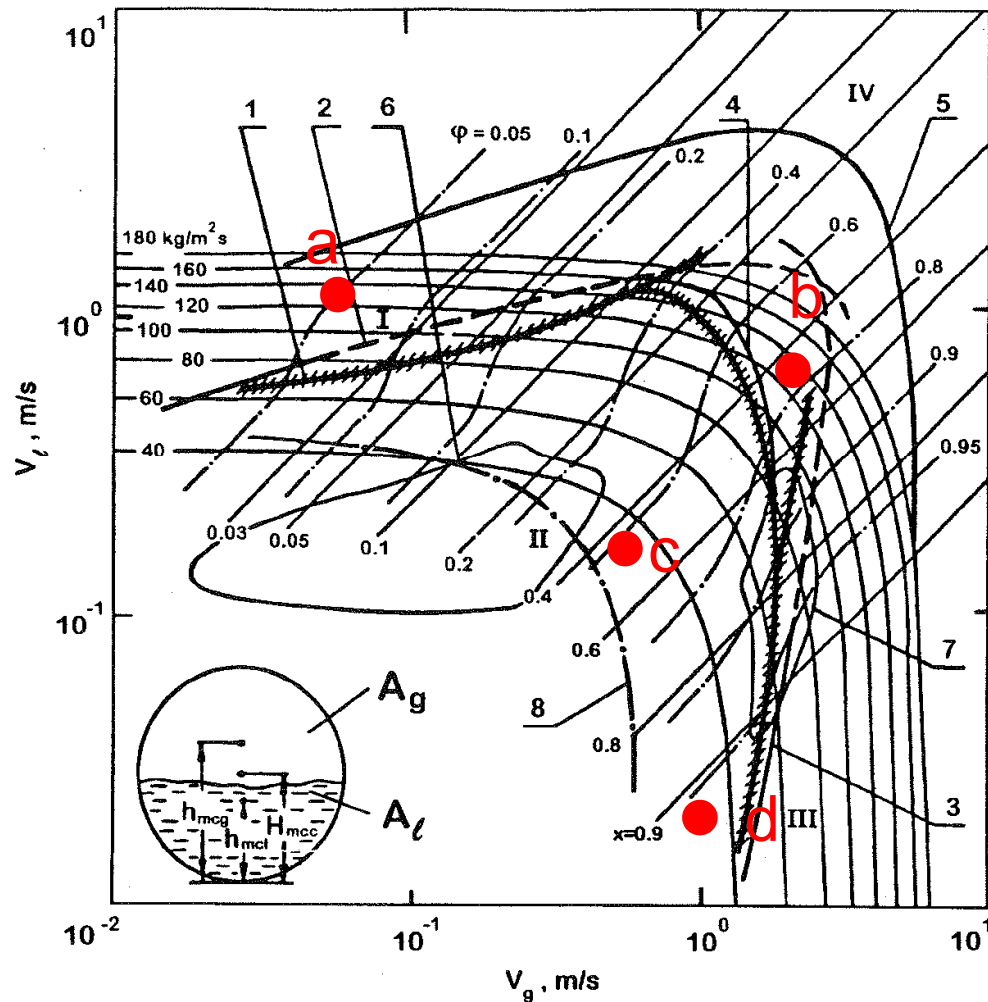
experimental boundaries for a pipe with inner diameter 7.9 mm

$$V_g = \frac{mx}{A\rho_g}$$

$$V_l = \frac{m(1-x)}{A\rho_l}$$

flow pattern map: $P = 70 \text{ W}$

Y. Filippov/Cryogenics 39 (1999) 59–68



Triangular cycle:

$P_{\text{total}} = 70 \text{ W}$

$P_{\text{coil}} = 19 \text{ W}$

$P_1 = 1.6 \text{ bar}$, $T_1 = 4.51 \text{ K}$

$m = 2.3 \text{ g/s}$

Coil:

a. $P = 1.5 \text{ bar}$, $x = 0.05$

b. $P = 1.1 \text{ bar}$, $x = 0.46$

Iron:

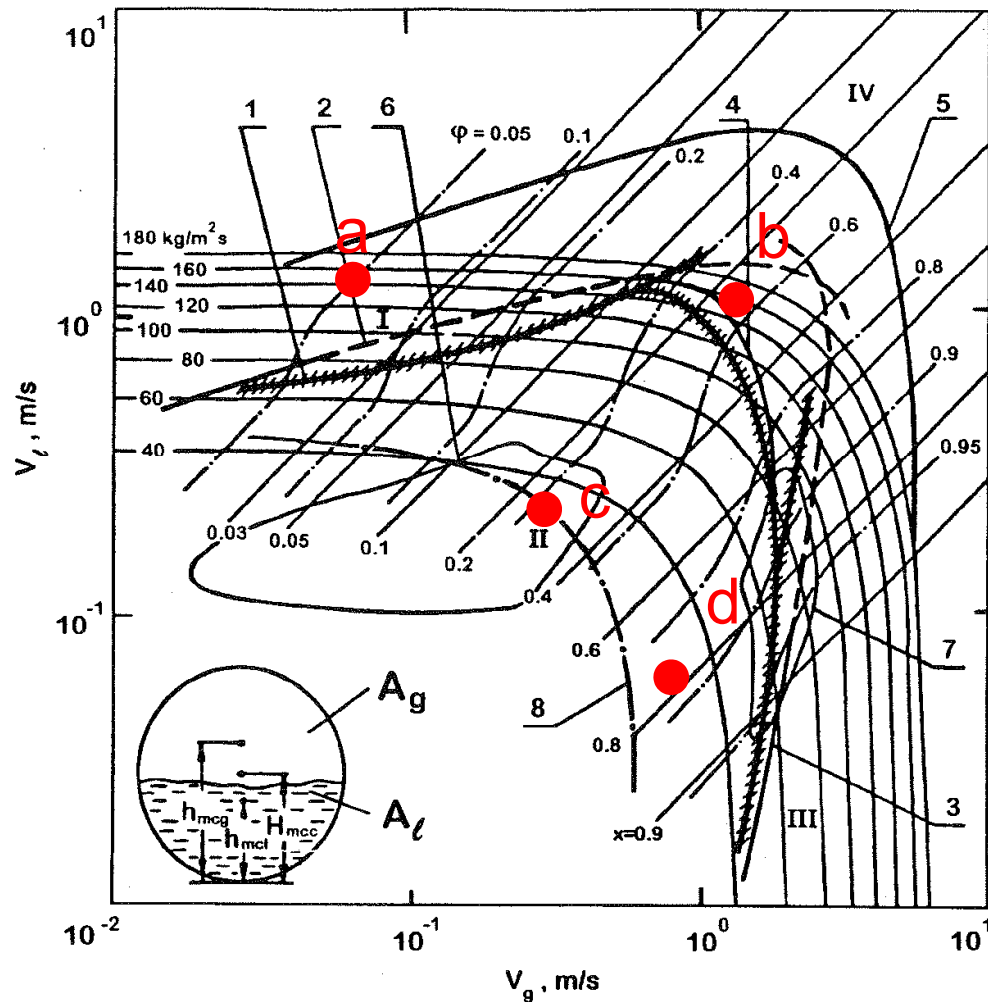
c. $P = 1.1 \text{ bar}$, $x = 0.46$

d. $P = 1.1 \text{ bar}$, $x = 0.95$

$P_3 = 1.1 \text{ bar}$, $T_3 = 5.91 \text{ K}$

flow pattern map: $P = 40 \text{ W}$

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Cycle 2c:
 $P_{\text{total}} = 40 \text{ W}$
 $P_{\text{coil}} = 10 \text{ W}$

$P_1 = 1.6 \text{ bar}$, $T_1 = 4.51 \text{ K}$
 $m = 2.7 \text{ g/s}$

Coil:

- a. $P = 1.5 \text{ bar}$, $x = 0.05$
- b. $P = 1.1 \text{ bar}$, $x = 0.24$

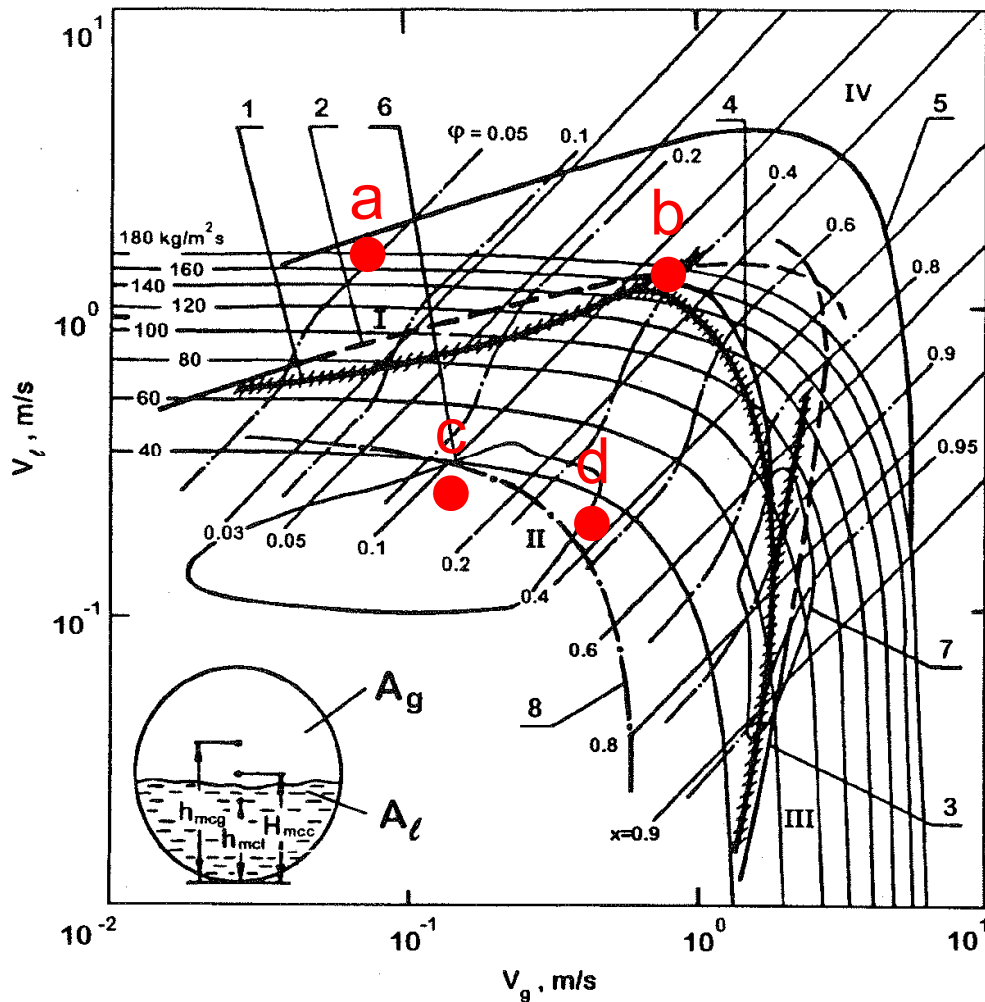
Iron:

- c. $P = 1.1 \text{ bar}$, $x = 0.24$
- d. $P = 1.1 \text{ bar}$, $x = 0.79$

$x_3 = 0.79$

flow pattern map: $P = 20 \text{ W}$

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$P_{\text{total}} = 20 \text{ W}$

$P_{\text{coil}} = 4 \text{ W}$

$P_1 = 1.6 \text{ bar}, T_1 = 4.51 \text{ K}$

$m = 3.0 \text{ g/s}$

Coil:

a. $P = 1.5 \text{ bar}, x = 0.1$

b. $P = 1.1 \text{ bar}, x = 0.12$

Iron:

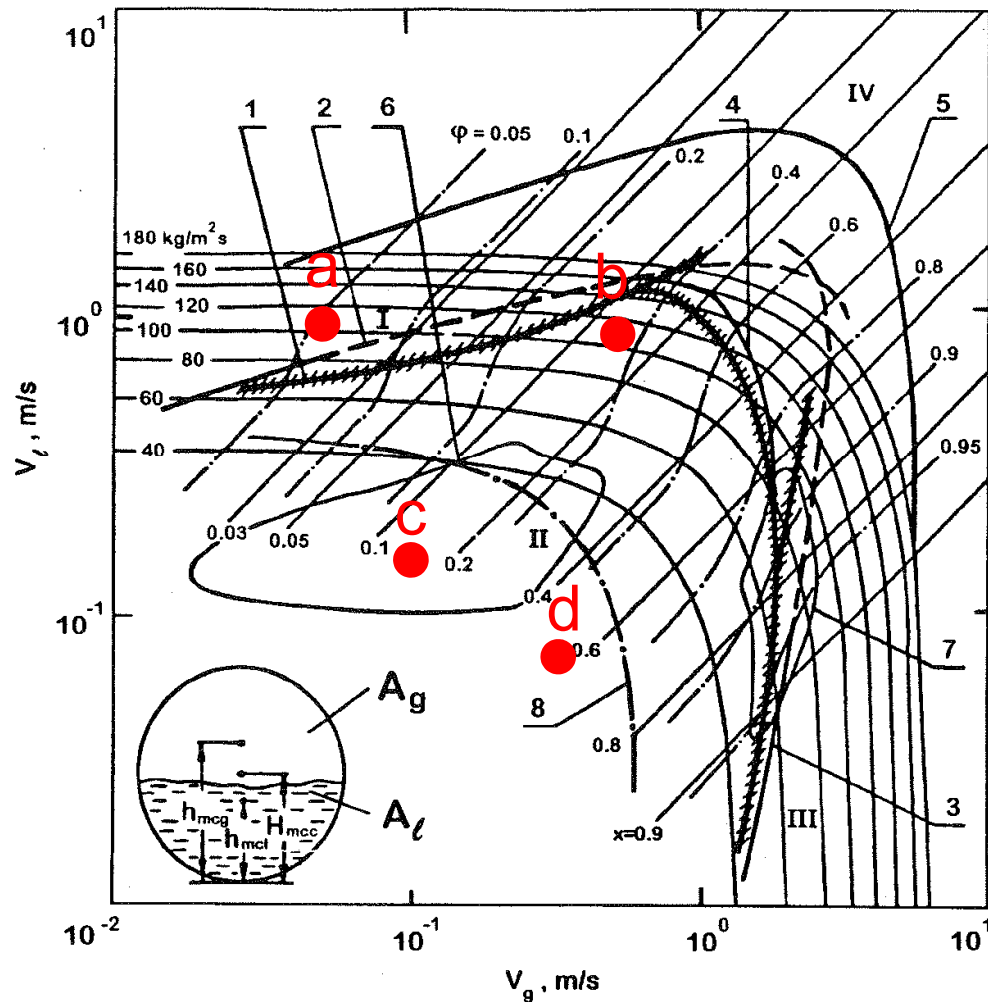
c. $P = 1.1 \text{ bar}, x = 0.12$

d. $P = 1.1 \text{ bar}, x = 0.39$

$x_3 = 0.39$

flow pattern map: $P = 20 \text{ W}$, $P_{in} = 1.35 \text{ bar}$

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$P_{total} = 20 \text{ W}$
 $P_{coil} = 4 \text{ W}$

$P_1 = 1.35 \text{ bar}$, $T_1 = 4.51 \text{ K}$
 $m = 1.9 \text{ g/s}$

Coil:

- a. $P = 1.5 \text{ bar}$, $x = 0.05$
- b. $P = 1.1 \text{ bar}$, $x = 0.16$

Iron:

- c. $P = 1.1 \text{ bar}$, $x = 0.16$
- d. $P = 1.1 \text{ bar}$, $x = 0.58$

$x_3 = 0.58$



Two-Phase Cooling

Potential problems:

- departure from nucleate boiling:
not expected because of very small heat flux
- stratified flow patterns:
dangerous only when occurs in the coil
- instabilities of two-phase flow in vertical channels:
should be small ($\rho_L/\rho_V \approx 7$)
- instabilities due to parallel channels

Arrangements needed for parallel cooling:

subcooled helium in the supply header

bypass valve in the end box

adjustments of hydraulic resistance for each channel

Adjustement of hydraulic resistances

Sector 1:

18 dipoles
14 quadrupole doublets
1 connection cryostat
(missing dipole)
47 parallel channels

quadrupole modules:

each doublet has 2 parallel flow branches

- Q (6 units in sector)
- Q + Steerer
- Q + Chr. Sextupole

Multipole Corrector cooled separately

Magnet Type	Sector					
	1	2	3	4	5	6
Dipole	18	18	18	18	18	18
Q + ST +Q	3	3	3	3	2	3
MC + Q + ST +Q	2	2	2	2	0	2
Q + ST + Q + CV (CH)	8	8	8	8	8	8
Connection Cryostat	2	2	2	2	2	2

Nuclotron Cooling vs. SIS100 cooling

Nuclotron:

- 2 sectors
- 100 parallel channels / sector
- 4.0 mm cable inner diameter
- beam vacuum chamber cooled by yoke

SIS100:

- 6 sectors
- 47 parallel channels / sector
- 4.7 mm cable inner diameter
- beam vacuum chamber cooled by helium flow

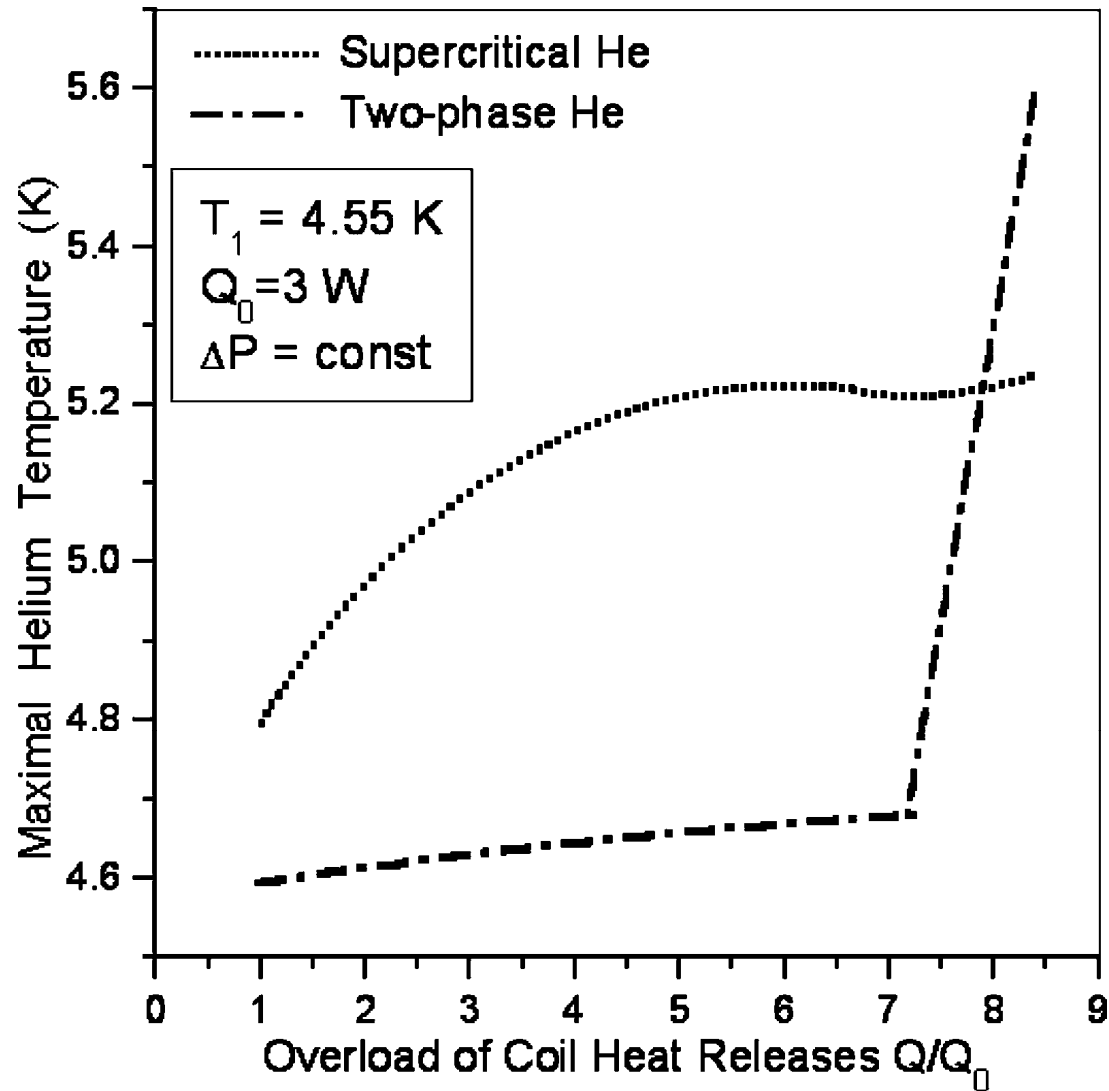


Conclusions

- SIS100 cooling systems is based on Nuclotron design
- the number of parallel channels is reduced by factor of two
- the phase separation in the coil is not expected
- adjustment of hydraulic resistance has to be performed for each type of cooling channel
- additional valve can be installed on each channel for fine adjustment



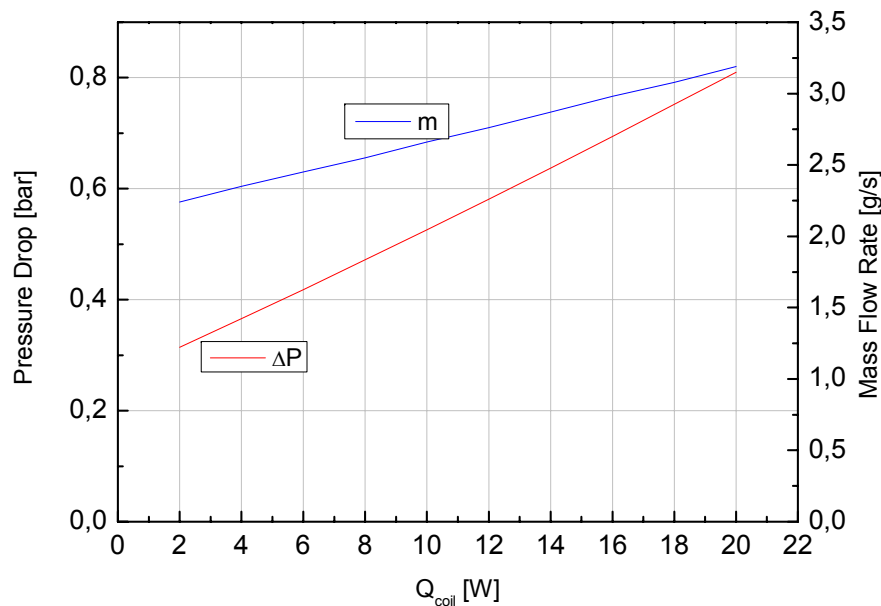
Two Phase vs. Supercritical



Two-Phase Cooling

$$Q_{\text{yoke}} = 30 \text{ W}$$

$$T_1 = 4.55 \text{ K}$$

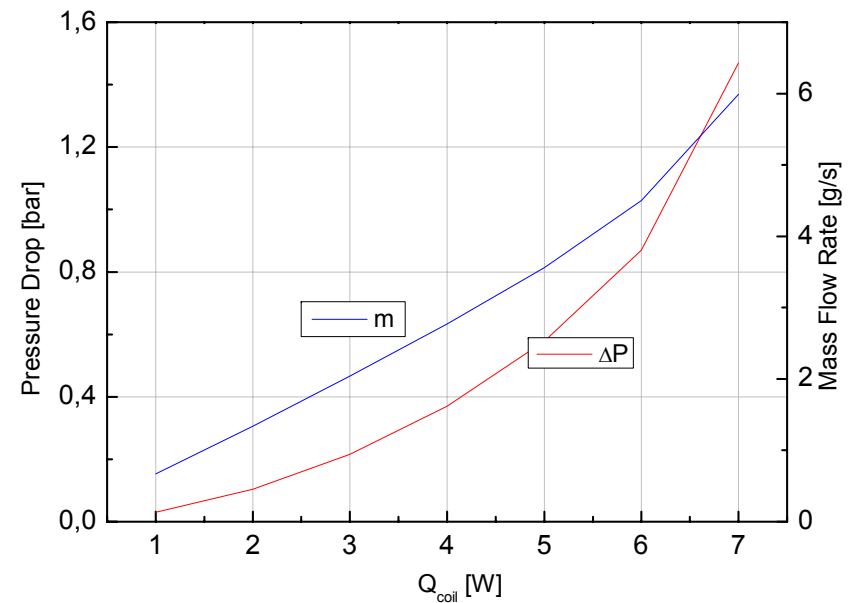


Two-Phase Helium:

$$P_2 = 1.1 \text{ bar}$$

$$X_3 = 1$$

$$m = (Q_{\text{coil}} + Q_{\text{yoke}}) / (h_3 - h_1)$$



Supercritical Helium:

$$P_2 = 2.3 \text{ bar}$$

$$T_3 = 4.7 \text{ K}$$

$$m = Q_{\text{coil}} / (h_2 - h_1)$$

Hydraulic Resistance

